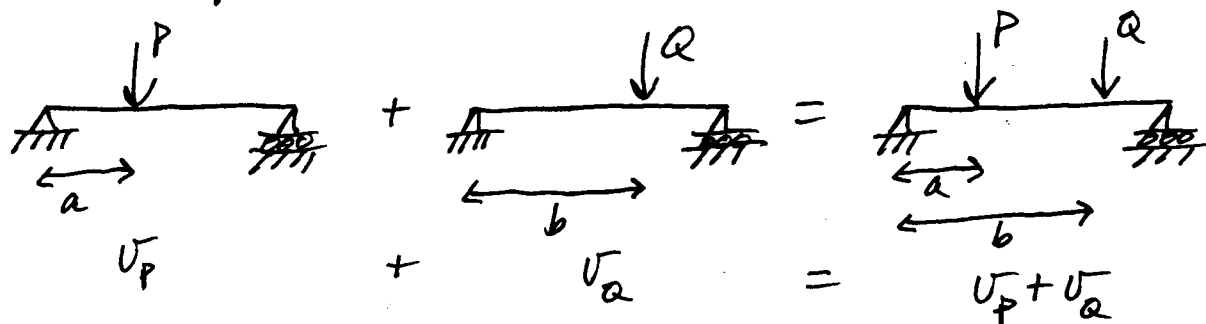


Beam Deflections using Superposition

The idea behind superposition is quite simple. Since $q = -\frac{1}{EI} \frac{d^4v}{dx^4}$ is a linear equation then if v_1 is a solution to $q_1 = -\frac{1}{EI} v_1''''$ and v_2 is a solution to $q_2 = -\frac{1}{EI} v_2''''$ then $v_1 + v_2$ is a solution to $q_1 + q_2 = -\frac{1}{EI} (v_1 + v_2)''''$. Note that this is useful when each of these problem solutions fits the same boundary conditions.

For example:



Let's generate the solution for an arbitrarily located point load on a simply supported beam again.

$$\begin{aligned}
 q(x) &= P \langle x-a \rangle^{-1} \\
 v(x) &= -P \langle x-a \rangle^0 + \frac{P(L-a)}{L} \\
 M(x) &= -P \langle x-a \rangle^1 + \frac{P(L-a)}{L} x \\
 \theta(x) &= -\frac{P}{2EI} \langle x-a \rangle^2 + \frac{P(L-a)}{2EIL} x^2 + C_1 \\
 v(x) &= -\frac{P}{6EI} \langle x-a \rangle^3 + \frac{P(L-a)}{6EIL} x^3 + C_1 x
 \end{aligned}$$

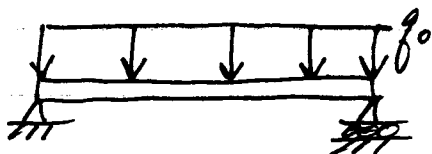
(121)

$$v(x=L) = 0 = -\frac{P}{6EI} \underbrace{(L-a)^3}_{L^3 - 3L^2a + 3La^2 - a^3} + \frac{P(L-a)}{6EI} L^2 + C_1 L$$

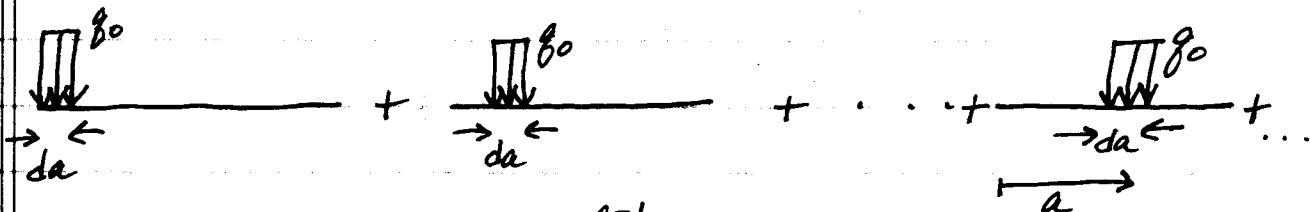
$$C_1 L = \frac{P}{6EI} (-2L^2a + 3La^2 - a^3)$$

$$v(x) = \frac{P}{6EI} \left[-\langle x-a \rangle^3 + \frac{L-a}{L} x^3 + \frac{a}{L} (2L^2 + 3La - a^2) x \right]$$

Can we use this solution along with superposition to find the deflection for a uniform $f(x) = q_0$?



This distribution looks like:

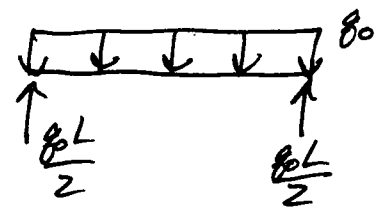


$$\text{Superposition} \rightarrow v(x) = \int_{a=0}^{a=L} \frac{q_0 da}{6EI} \left[-\langle x-a \rangle^3 + \frac{L-a}{L} x^3 + \frac{a}{L} (2L^2 + 3La - a^2) x \right]$$

$$\begin{aligned} v(x) &= \int_{a=0}^{a=x} \frac{-q_0}{6EI} (x-a)^3 da + \int_{a=0}^{a=L} \frac{q_0}{6EI} \left[\left(1 - \frac{a}{L}\right) x^3 + \frac{1}{L} (2L^2 + 3La - a^2) x \right] da \\ &= \frac{q_0}{24EI} (x-a)^4 \Big|_{a=0}^{a=x} + \frac{q_0}{6EI} \left\{ Lx^3 - \frac{L}{2} x^3 - L^3 x + L^3 x - \frac{L^3}{4} x \right\} \end{aligned}$$

$$v(x) = \frac{-q_0}{24EI} x^4 + \frac{q_0}{12EI} x^3 - \frac{q_0 L^3}{24EI} x$$

Is this correct?



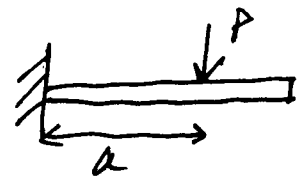
$$\begin{aligned}
 q(x) &= q_0 \\
 V(x) &= -q_0 x + \frac{q_0 L}{2} \\
 M(x) &= -\frac{q_0}{2} x^2 + \frac{q_0 L}{2} x \\
 \theta(x) &= -\frac{q_0}{6EI} x^3 + \frac{q_0 L}{4EI} x^2 + C_1
 \end{aligned}$$

$$v(x) = -\frac{q_0}{24EI} x^4 + \frac{q_0 L}{12EI} x^3 + C_1 x$$

$$v(x=L) = 0 = -\frac{q_0}{24EI} L^4 + \frac{q_0}{12EI} L^4 + C_1 L$$

$$\rightarrow C_1 = -\frac{q_0 L^3}{24EI} \quad \checkmark$$

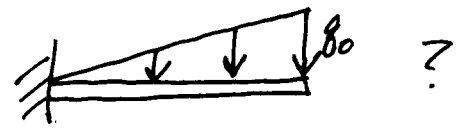
Used this way, the point force solution is called a Green's function. This function can be integrated to determine the solution for any distribution of $q(x)$ by using $P \rightarrow q(a) da$ as long as the beam is supported by simple supports on its ends. The Green's function for a cantilever support would be the solution to:



Following the steps on (120) yields

$$v(x) = -\frac{P}{6EI} \langle x-a \rangle^3 + \frac{P}{6EI} x^3 - \frac{Pa}{2EI} x^2$$

What is $v(x)$ for



$$f(x) = f_0 \frac{x}{L} \Rightarrow P \rightarrow f_0 \frac{a}{L} da$$

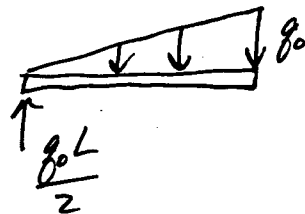
$$v(x) = \frac{1}{EI} \int_{a=0}^{a=L} -\frac{f_0 a}{6L} (x-a)^3 + \frac{f_0 a}{6L} x^3 - \frac{f_0 a^2}{2L} x^2 da$$

$$= \frac{f_0}{EIL} \left\{ \int_{a=0}^{a=x} -\frac{a}{6} (x-a)^3 da + \int_{a=0}^{a=L} \frac{a}{6} x^3 - \frac{a^2}{2} x^2 da \right\}$$

$$\quad \quad \quad -\frac{1}{6}(x^3 a - 3x^2 a^2 + 3x a^3 - a^4)$$

$$= \frac{f_0}{EIL} \left\{ \underbrace{\left[-\frac{1}{12} x^3 a^2 + \frac{1}{6} x^2 a^3 - \frac{1}{8} x a^4 + \frac{1}{30} a^5 \right]_{a=0}^{a=x}}_{-\frac{x^5}{120}} + \frac{L^2}{12} x^3 - \frac{L^3}{6} x^2 \right\}$$

$$v(x) = \frac{f_0}{EIL} \left(-\frac{x^5}{120} + \frac{L^2 x^3}{12} - \frac{L^3 x^2}{6} \right)$$

Is this correct? 

$$f(x) = f_0 \frac{x}{L}$$

$$V(x) = -\frac{f_0}{2L} x^2 + \frac{f_0 L}{2}$$

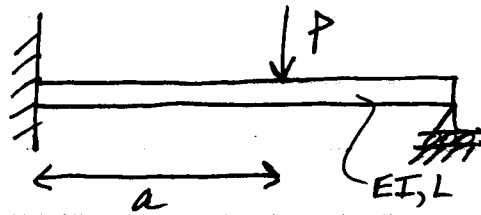
$$M(x) = -\frac{f_0}{6L} x^3 + \frac{f_0 L}{2} x - \frac{f_0 L^2}{3}$$

$$\theta(x) = -\frac{f_0}{24EI} x^4 + \frac{f_0 L}{4EI} x^2 - \frac{f_0 L^2}{3EI} x$$

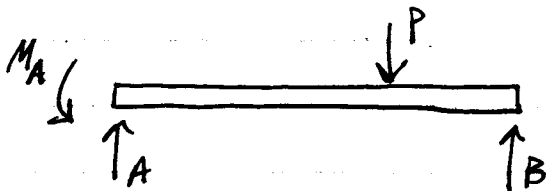
$$v(x) = -\frac{f_0}{120EI} x^5 + \frac{f_0 L}{12EI} x^3 - \frac{f_0 L^2}{6EI} x^2 \quad \checkmark$$

This is just one use of superposition. Another important application is to statically indeterminate problems.

Consider the problem:



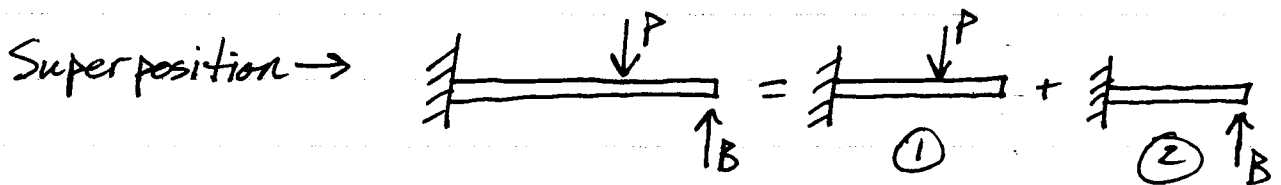
What are the reactions,
and what is the deflection
of the beam?



$$\sum F_y = A + B - P = 0$$

$$\sum M_z^A = M_A + BL - Pa = 0$$

Not enough equations for
unknowns \rightarrow statically indeterminate.



as we just found on page (122),

$$v_1(x) = \frac{-P}{6EI} \langle x-a \rangle^3 + \frac{P}{6EI} x^3 - \frac{Pa}{2EI} x^2$$

The same formula can be used for $v_2(x)$.

$$v_2(x) = \frac{-B}{6EI} x^3 + \frac{BL}{2EI} x^2$$

$$\text{So, } v(x) = v_1(x) + v_2(x) = \frac{-P}{6EI} \langle x-a \rangle^3 + \frac{P-B}{6EI} x^3 + \frac{BL-Pa}{2EI} x^2$$

Now, we know P , but B is a reaction. How
can we determine B ?

We have the constraint that $v(x=L) = 0$.

$$v(L) = -\frac{P}{6EI} (L-a)^3 + \frac{P-B}{6EI} L^3 + \frac{BL-Pa}{2EI} L^2 = 0$$

$$B = \frac{3P}{L^3} \left[\frac{1}{6} (L-a)^3 - \frac{1}{6} L^3 + \frac{a}{2} L^2 \right]$$

$$B = 3P \left[\frac{1}{6} \left(1 - \frac{a}{L}\right)^3 - \frac{1}{6} + \frac{1}{2} \frac{a}{L} \right]$$

Checks: $a=0 \rightarrow B=0$ $B = 3P \left[\frac{1}{6} - \frac{1}{6} \right] \checkmark$
 $a=L \rightarrow B=P$ $B = 3P \left[-\frac{1}{6} + \frac{1}{2} \right] = P \checkmark$

Now that we have B we know $v(x)$. Also, our equilibrium equations tell us A and M_A .

$$A = P - B$$

$$M_A = Pa - BL$$

We could also get A and M_A from $v(x)$.

$$A = V(x=0) = EI v''' \Big|_{x=0}$$

$$M_A = -M(x=0) = -EI v'' \Big|_{x=0}$$

Another approach: $q(x) = P(x-a)^{-1}$

$\rightarrow V(x) = -P(x-a)^0 + A$
 $\uparrow V(0) = A$

$M(x) = -P(x-a)^1 + Ax - M_A$
 $\uparrow M(0) = -M_A$

$\theta(x) = -\frac{P}{2EI}(x-a)^2 + \frac{A}{2EI}x^2 - \frac{M_A}{EI}x + 0$
 $\uparrow \theta(x=0) = 0$

$v(x) = -\frac{P}{6EI}(x-a)^3 + \frac{A}{6EI}x^3 - \frac{M_A}{2EI}x^2 + 0$
 $\uparrow v(0) = 0$

Boundary conditions at $x=L$:

$v(L) = 0 = -\frac{P}{6EI}(L-a)^3 + \frac{A}{6EI}L^3 - \frac{M_A}{2EI}L^2$

$M(L) = EI v''(L) = 0 = -P(L-a) + AL - M_A$

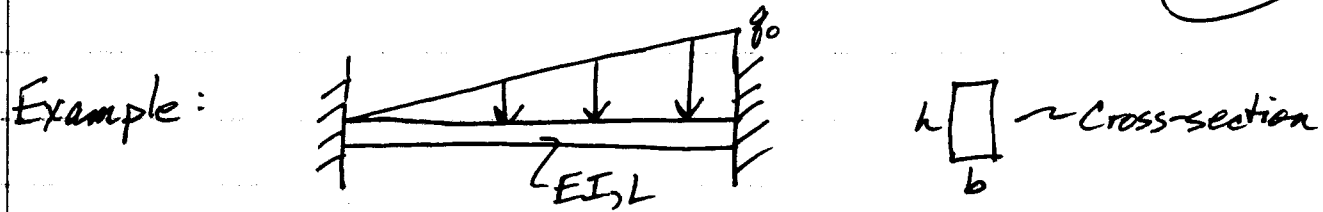
Solve for $M_A \neq A \rightarrow M_A = AL - P(L-a)$

then $0 = -\frac{1}{3}P(L-a)^3 + \frac{1}{3}L^3A - L^3A + PL^2(L-a)$

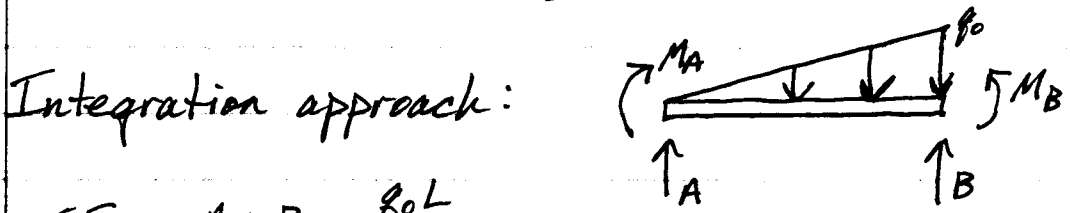
$A = -\frac{1}{2}P(1-\frac{a}{L})^3 + \frac{3}{2}P(1-\frac{a}{L})$

From equilibrium $B = P - A = \frac{1}{2}P(1-\frac{a}{L})^3 - \frac{1}{2}P + \frac{3}{2}P\frac{a}{L}$

✓ this result is the same as our previous approach.



Determine τ_{max} , σ_{max} , v_{max} .



$$\sum F_y = A + B - \frac{q_0 L}{2} = 0$$

$$\sum M_z = -M_A + M_B + BL - \frac{q_0 L}{2} \frac{2}{3} L = 0$$

2 equations for 4 unknowns

$$q(x) = q_0 \frac{x}{L}$$

$$V(x) = -\frac{q_0}{2L} x^2 + \frac{A}{L} \quad \text{from } V(0) = A$$

$$M(x) = -\frac{q_0}{6L} x^3 + Ax + \frac{M_A}{L} \quad \text{from } M(0) = M_A$$

$$\theta(x) = -\frac{q_0}{24EI L} x^4 + \frac{A}{2EI} x^2 + \frac{M_A}{EI} x \quad \text{using } \theta(0) = 0$$

$$v(x) = -\frac{q_0}{120EI L} x^5 + \frac{A}{6EI} x^3 + \frac{M_A}{2EI} x^2 \quad \text{using } v(0) = 0$$

Now we need boundary conditions at $x=L$.

$$v(L) = 0 = -\frac{q_0 L^5}{120EI L} + \frac{AL^3}{6EI} + \frac{MAL^2}{2EI}$$

$$\theta(L) = 0 = -\frac{q_0 L^4}{24EI L} + \frac{AL^2}{2EI} + \frac{MAL}{EI}$$

Solve for A & M_A .

$$M_A = \frac{q_0 L^2}{24} - \frac{AL}{2}$$

$$\rightarrow 0 = \frac{-q_0 L^2}{60} + \frac{AL}{3} + \frac{q_0 L^2}{24} - \frac{AL}{2}$$

$$\frac{A}{6} = \frac{q_0 L}{40} \rightarrow A = \frac{3q_0 L}{20}$$

$$\rightarrow M_A = \frac{-4q_0 L^2}{120}$$

Equilibrium $\rightarrow B = \frac{7}{20} q_0 L$

$$M_B = \frac{-16}{120} q_0 L^2$$

Do the signs make sense? (Discuss)

Check B & M_B.

$$V(L) = -B \rightarrow \frac{-7}{20} q_0 L \stackrel{?}{=} -\frac{q_0}{2L} L^2 + \frac{3q_0 L}{20} = \frac{-7}{20} q_0 L$$

$$M(L) = M_B \rightarrow \frac{-16}{120} q_0 L^2 \stackrel{?}{=} -\frac{q_0 L^2}{6} + \frac{3q_0 L^2}{20} - \frac{4q_0 L^2}{120} = \frac{-5}{120} q_0 L^2$$

$$U(x) = -\frac{q_0}{120EI} x^5 + \frac{q_0 L}{40EI} x^3 - \frac{4q_0 L^2}{240EI} x^2$$

$$U_{max} \text{ occurs where } \theta = 0 \rightarrow \frac{-q_0}{24EI} x_c^4 + \frac{3q_0 L}{40EI} x_c^2 - \frac{4q_0 L^2}{120EI} x_c = 0$$

$x_c = 0$ is a solution & we should get $x_c = L$ as well.

$$\text{Factor } x_c = 0 \text{ out} \rightarrow \frac{-1L^2}{24L^3} x_c^3 + \frac{3L^2}{40L} x_c - \frac{4L^2}{120} = 0$$

Factor $\frac{x_c}{L} = 1$ out ~~scribbled out~~

$$\left(\frac{x_c}{L} - 1\right) \left[-\frac{1}{24} \left(\frac{x_c}{L}\right)^2 - \frac{1}{24} \left(\frac{x_c}{L}\right) + \frac{1}{30}\right] = 0$$

$$\frac{x_c}{L} = \frac{\frac{1}{24} \pm \sqrt{\left(\frac{1}{24}\right)^2 + \frac{1}{180}}}{-2/24}$$

$$\frac{x_c}{L} = -\frac{1}{2} + \left(\sqrt{1 + \frac{576}{180}} / 2\right) \approx 0.525$$

$$\rightarrow V_{\max} = 0.0013085 \frac{q_0 L^4}{EI}$$

V_{\max} occurs at one of the supports since $q \neq 0$ in the range $0 < x < L$.

$$B > A \rightarrow V_{\max} = \frac{7}{20} q_0 L \quad (\text{Note that here it is OK to deal with magnitudes.})$$

$$I_{\max} = \frac{V_{\max}}{I} \left(\frac{Q}{b}\right)_{\max}$$

For a rectangle Q_{\max} occurs at $y=0$.



$$Q(y=0) = \int_0^{h/2} y' b dy' = \left. \frac{b}{2} y'^2 \right|_0^{h/2} = \frac{bh^2}{8}$$

$$\rightarrow I_{\max} = \frac{7}{20} q_0 L \frac{12}{bh^3} \frac{bh^2}{8b} = \frac{21}{40} \frac{q_0 L}{bh}$$

M_{\max} occurs where $V = 0$.

$$V = -\frac{q_0}{2L} x_0^2 + \frac{3q_0L}{20} = 0$$

$$x_0 = \sqrt{\frac{3}{10}} L$$

$$M\left(\sqrt{\frac{3}{10}}L\right) = q_0L^2 \left[-\frac{1}{6} \left(\sqrt{\frac{3}{10}}\right)^3 + \frac{3}{20} \sqrt{\frac{3}{10}} - \frac{1}{30} \right]$$

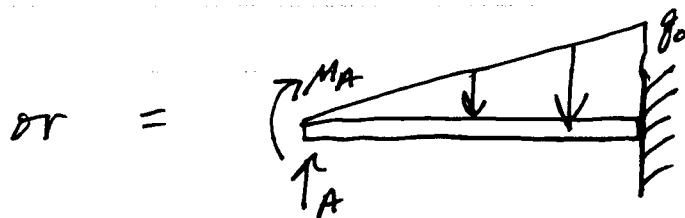
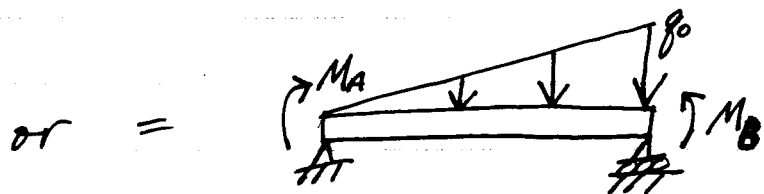
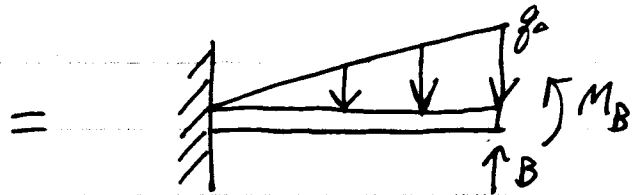
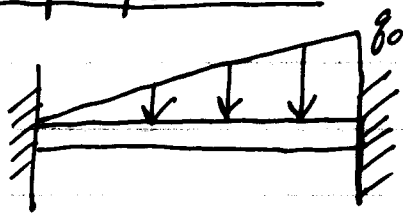
$$= 0.0214 q_0L^2$$

$$M_A = -\frac{1}{30} q_0L^2 = -0.033 q_0L^2$$

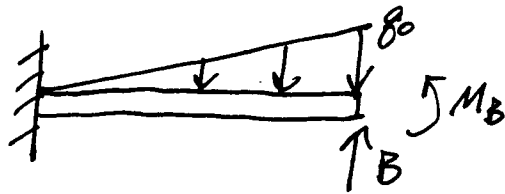
$$M_B = -\frac{1}{20} q_0L^2 = \boxed{-0.05 q_0L^2} \text{ max magnitude}$$

$$\sigma_{max} = \frac{M_B h/2}{bh^3/12} = \frac{3}{10} \frac{q_0L^2}{bh^3} \quad \left(\begin{array}{c} \text{tens} \\ \text{comp} \end{array} \right)$$

Superposition



Any of these can be used for a superposition scheme. Let's look at the first.



$$= \text{[Diagram: beam with triangular load } g_0] \Rightarrow v_g = \frac{-g_0 x^2}{120EI} (20L^3 - 10L^2x + x^3)$$

$$+ \text{[Diagram: beam with reaction force } B] \Rightarrow v_B = \frac{Bx^2}{6EI} (3L - x)$$

$$+ \text{[Diagram: beam with reaction moment } M_B] \Rightarrow v_M = \frac{M_B x^2}{2EI}$$

$$v(x) = v_g + v_B + v_M = \frac{-g_0 x^2}{120EI} (20L^3 - 10L^2x + x^3) + \frac{Bx^2}{6EI} (3L - x) + \frac{M_B x^2}{2EI}$$

Boundary conditions:

$$v(L) = 0 = \frac{-g_0 L^2}{120EI} \underbrace{(20 - 10 + 1)}_{11} + \frac{BL^3}{6EI} \underbrace{(3 - 1)}_2 + \frac{M_B L^2}{2EI}$$

$$\theta(L) = v'(L) = 0 = -\frac{g_0 L^3}{120EI} \underbrace{(40 - 30 + 5)}_{15} + \frac{BL^2}{6EI} \underbrace{(6 - 3)}_3 + \frac{M_B L}{EI}$$

$$M_B = \frac{15}{120} g_0 L^2 - \frac{BL}{2}$$

$$0 = -\frac{11}{120} g_0 L^2 + \frac{BL}{3} - \frac{BL}{4} + \frac{15}{240} g_0 L^2$$

$$\rightarrow B = \frac{7}{20} g_0 L \quad \checkmark \quad \text{same as previous approach}$$

Use $\delta = -EIv''''$, $V = EIv'''$, $M = EIv''$ to find max/min.